

A simple analytical model for pressure on obstacles induced by snow avalanches

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ABSTRACT. The forces snow avalanches are able to exert on protection dams or buildings is of crucial interest in order to improve avalanche mitigation measures and to quantify the mechanical vulnerability of structures likely to be damaged by snow avalanches. This paper presents an analytical model that is able to calculate these forces taking into account dead zone mechanisms. First, we present a 2D analytical hydrodynamic model describing the forces on a wall overflowed by gravity-driven flows down an inclined plane. Second, the 2D model is successfully validated on discrete simulations of granular flows. Third, we provide ingredients to extend the 2D model to flows of dry and cold snow. Fourth, we propose a simplified 3D analytical model taking into account lateral fluxes. Finally, the predictions from the simplified 3D analytical model are successfully compared to recent measurements on two full-scale snow avalanches released at the Lautaret site in France.

INTRODUCTION

The influence of obstacles on avalanche flows has been the topic of many recent studies combining full-scale observations on snow avalanches, small-scale experiments with granular materials, theory and numerical modelling. Studies refer to avalanche flows interacting with deflecting dams (Irgens and others, 1998; Jóhannesson, 2001; Hákonardóttir and Hogg, 2005; Cui and others, 2007; Gray and Cui, 2007; Faug and others, 2007), catching dams (Chu and others, 1995; McClung and Mears, 1995;

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23 Naaim and others, 2004; Faug and others, 2008a,b; Gauer and others, 2007, 2009) and retarding mounds (Hákonardóttir and
24 others, 2003; Chiou and others, 2005). Up to now the particular situation of free-surface flows overflowing a catching dam
25 has been addressed in terms of the runout shortening downstream of the dam (Faug and others, 2008a,b). This first question
26 is of crucial interest with regard to the residual risk downstream of protection dams. This paper deals with the force these
27 avalanche flows are able to exert on a flat obstacle when overflowing it. This second question is also important in order to
28 estimate the mechanical vulnerability of buildings and protection dams. Recent full-scale observations on snow avalanches
29 report large pressures at low incoming Froude numbers on tubular pylons (Sovilla and others, 2008a,b) and on a flat obstacle
30 - a 1 m² plate - (Thibert and others, 2008; Sovilla and others, 2008a). Complex materials such as granular materials or dry
31 snow can behave as a fluid or a solid: their ability to undergo a transition from a fluid state to a solid state can lead to the
32 formation of stagnant zones when the flow encounters a topographic discontinuity of the ground. While a part of the incoming
33 material is jammed (solid-like phase) another part of the material remains in a fluid-like phase and is deflected, being able
34 to overflow the obstacle. The ability of a granular fluid to form stagnant zones, or so called granular “dead zones” (Faug and
35 others, 2002; Gray and others, 2003; Faug and others, 2008a), upstream of a topographic discontinuity may lead to severe
36 modifications of the resulting force on the obstacle. The size of the stagnant zone formed upstream of a catching dam is very
37 large if the catching dam prevents side flows. In this paper, we present a simple analytical model based on momentum balance
38 over a control volume upstream a flat obstacle in order to analyse and quantify the effect of the dead zone mechanism on
39 the resulting force. The analytical model is first developed for 2D flows and validated in a discrete numerical model for dry
40 granular flows overflowing a dam. Asymptotic behaviours at low and high Froude numbers are discussed. Then we propose to
41 extrapolate the model to 2D dry snow flows using a Voellmy friction law. Third, the model is adapted to a more complex 3D
42 geometry taking into account lateral fluxes. Finally we compare the predictions of a 3D simplified analytical model to the field
43 data available from a 1 m² plate subject to the impact of snow avalanches, at the Lautaret test site in France. It is shown that
44 our analytical model is able to properly reproduce the measured force values within the experimental uncertainty mentioned
45 by the related previous studies (Thibert and others, 2008; Baroudi and Thibert, 2009).

46 **MODELLING OF FREE-SURFACE FLOWS OVERFLOWING A DAM**

47 **General framework equations in 2D geometry**

48 We consider a steady incoming free-surface flow of thickness h , mean velocity u and mean density ρ down a flat slope with
49 inclination θ as shown in Figure 1. A nearly triangular stagnant zone is formed upstream of the dam as shown in Figure 1.
50 Momentum conservation over a control volume V_0 upstream of the dam (pink colored area in Fig. 1) allows us to show that,
51 in a steady flow regime, the resulting force F normal to the dam is the sum of four contributions:

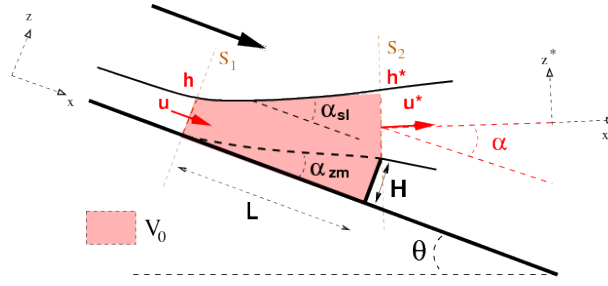


Fig. 1. Sketch of the flow and control volume V_0 (pink colored area). H is the obstacle height. α_{zm} is the mean angle of the dead zone, α_{sl} is the mean angle of the free-surface above the dead zone, α is the angle between the velocity \vec{u}^* and the ground. h and u are the thickness and the depth averaged velocity outside the influence area of the obstacle. h^* and u^* are the thickness and the depth averaged velocity of the overflow at the top of the dam (in x^* direction). L is the length of the influence zone upstream of the obstacle assumed to be close to the length of the dead zone.

$$F = F_u + F_h + F_w + F_f \quad (1)$$

V_0 is a volume per unit width and we consider forces per unit width. F_u is a purely dynamic force resulting from the momentum variation between sections S_1 and S_2 defined in Figure 1: $F_u = \beta(1 - \delta_u \cos \alpha) \rho u^2 h$, where δ_u is the velocity ratio u^*/u , u^* is the velocity at the top of the dam as defined in Figure 1 and α is the deflection angle (i.e the angle between \vec{u}^* and the ground). The coefficient β depends on the shape of the velocity profile and is defined by $\frac{1}{h} \int_0^h u^2 dz = \beta (\frac{1}{h} \int_0^h u dz)^2$. The relative velocity reduction $(u - u^*)/u$ is simply assumed to be proportional to the deflection angle α , which gives:

$$\delta_u = 1 - \kappa \alpha \quad (2)$$

where κ is a velocity reduction coefficient defined later in the paper. F_h is a purely hydrostatic contribution due to the incoming flow undisturbed by the obstacle $F_h = \frac{1}{2} k \rho g h^2 \cos \theta$, where k is the earth pressure coefficient classically introduced for gravity-driven flows of granular materials (Savage and Hutter, 1989) or snow (McClung and Mears, 1995; Bartelt and others, 1999). F_w is the x -axis component of the weight of the control volume V_0 : $F_w = \rho g V_0 \sin \theta$. F_f is the basal friction force assumed to be proportional to the y -axis component of the weight of the control volume V_0 (Coulomb friction): $F_f = \mu_{zm} \rho g V_0 \cos \theta$, where μ_{zm} is the friction coefficient between the dead zone and the ground. We will define μ_{zm} later in the paper.

The volume V_0 can be calculated from Figure 1 as follows:

$$V_0 \approx \frac{1}{2} \left[\left(H + h \left(1 + \frac{\delta_h}{\cos \alpha} \right) \right) L - h^2 \delta_h^2 \tan \alpha \right] \quad (3)$$

64 where δ_h is the depth ratio equal to h^*/h with h^* being the thickness of the outcoming flow defined in Figure 1, H is the
 65 obstacle height and L is the length of the influence zone upstream the obstacle. The zone of influence upstream the dam is
 66 defined as the length of the disturbed flow upstream of the dam (flow depth and velocity are not equal to the incoming flow
 67 depth h and velocity u) and can be approximated by: $\tan(\alpha_{zm}) = H/L$. By mass flow rate conservation, the flow depth ratio
 68 δ_h is equal to $1/\delta_u$ if we assume that the density is unchanged. The angle α is equal to $(\alpha_{sl} + \alpha_{zm})/2$ where α_{sl} is the angle
 69 of the free-surface inside the control volume V_0 . We will detail later on how we can estimate the angles α_{zm} and α_{sl} .

70 If we neglect the second-order term $[\delta_h^2 \tan \alpha]$ in equation 3, equation 1 can be synthesized in terms of the normalized force
 71 $F/(0.5\rho u^2 h)$ versus the incoming Froude number $Fr = u/\sqrt{gh \cos \theta}$:

$$\frac{F}{\frac{1}{2}\rho u^2 h} = 2\beta(1 - (1 - \kappa\alpha) \cos \alpha) + \frac{1}{Fr^2} \left[k + \left(\frac{\sin \theta - \mu_{zm} \cos \theta}{\tan \alpha_{zm}} \right) \left(\frac{H}{h} + 1 + \frac{1}{(1 - \kappa\alpha) \cos \alpha} \right) \frac{H}{h} \right] \quad (4)$$

72 The second-order term $[\delta_h^2 \tan \alpha]$ is strickly negligible when $h^* \ll L$ and $h^* \approx h$, which is almost true when the Froude
 73 number is not too high (typically less than 5 – 10). When the Froude number is more than 5 – 10, the contribution due to the
 74 volume V_0 in the total force is so weak (purely dynamic force) that an error in V_0 has no effect on the total resulting force.
 75 This analytical model has been initially developed for granular flows (Faug and others, 2009) for which the parameters (k , β ,
 76 μ_{zm} , κ) have been determined as well as expressions to calculate the angles α_{zm} and α_{sl} , which give the deflection angle α .
 77 Results for granular flows are briefly reported in the next section.

78 Validation of the analytical model for 2D granular flows

79 We performed sphere discrete particle simulations using a linear damped spring law between particles with a Coulomb failure
 80 criterion in order to simulate 2D steady granular flows down an inclined slope as shown in Figure 1 (see details in Faug and
 81 others, 2009). These discrete numerical simulations were based on the molecular dynamics method as introduced by Cundall
 82 and Strack (1979) and largely used to simulate dense granular flows (Silbert and others, 2001; Ertas and others, 2001; da Cruz
 83 and others, 2005). The following microscopic parameters were needed to describe the contacts between grains: the normal and
 84 tangential stiffnesses k_n and k_t , the restitution coefficient e (linked to the damping coefficient) and the interparticle friction
 85 coefficient μ . The influence of these parameters has been already discussed by Silbert and others (2001): (i) k_n and k_t have no
 86 effects in the limit of rigid grains (overlap between particles less than 1/1000), (ii) e has little effect except for extreme values
 87 $e = 0$ and $e = 1$, (iii) μ has a greater effect on the results but its influence becomes weak at low values typically smaller than
 88 0.5. In our simulations, we used the following values: $k_n = 10^4 \text{ N m}^{-1}$, $k_t = 1/2k_n$, $e = 0.5$ and $\mu = 0.5$ (corresponding to a
 89 typical value of the internal friction angle of granular materials). Here we used the commercial code PFC^{2D} (Itasca consulting:
 90 <http://www.itasca.com/pfc/index.php>). These numerical simulations allowed us to estimate the macroscopic empirical laws

to close Equation 4 for the case of granular materials. A first result of the numerical simulations is that the basal friction μ_{zm} was shown to be constant equal to $(\tan \theta_{min})$, where θ_{min} is the minimal angle below which no steady flow is possible (stopping of the flow) (Pouliquen, 1999; GDRMiDi, 2004). Second, the mean angle of the dead zone with the horizontal, $\theta - \alpha_{zm}$, was shown to be equal to θ_{min} for all slope inclinations θ . The length L of the influence zone of the obstacle was then assumed to be equal to the length of the stagnant zone and accordingly was defined as: $\tan(\theta - \theta_{min}) = H/L$. Third, the free-surface angle α_{sl} was shown to be a simple linear function of the slope inclination θ : $\alpha_{sl} = a\theta + b$, where a and b are coefficients depending on the incoming flow regime. We defined θ_{max} as the maximum angle above which uniform flows were not possible. For uniform flows ($\theta_{min} < \theta < \theta_{max}$), the angle α_{sl} was expressed as:

$$\alpha_{sl} = \frac{\theta_{min}}{\theta_{max} - \theta_{min}}(\theta - \theta_{min}) \quad (5)$$

For non uniform flows ($\theta > \theta_{max}$), the angle α_{sl} was expressed as:

$$\alpha_{sl} = \frac{\pi}{2} - \left(\frac{\theta_{min} - \pi/2}{\theta_{max} - \pi/2} \right) (\pi/2 - \theta) \quad (6)$$

Equation 4 was successfully tested on data from discrete numerical simulations using the following values (Faug and others, 2009): $k = 1$, $\beta = 5/4$ (for a Bagnold-like velocity profile (GDRMiDi, 2004)), $\kappa = (1 - e)/(\pi/2)$ (where $e = 0.5$ is the restitution coefficient), $\theta_{min} = 14^\circ$ and $\theta_{max} = 24^\circ$ (typical values for 2D granular flows (GDRMiDi, 2004)). The value of κ is simply estimated from a purely collisional regime assuming that u^*/u scales as e when $\alpha = \pi/2$ which gives $e = 1 - \kappa(\pi/2)$ according to Equation 2. Figure 2 shows predictions of the analytical model compared to discrete simulations. The predictions of our analytical model are in very good agreement with the discrete simulations without having introduced any fitting parameter. Note that Equation 4 predicts a force ratio $F/(0.5\rho u^2 h) = [2\beta(1 - (1 - \kappa\alpha) \cos \alpha)]$ at high Froude numbers, which gives a value of 2 for $\alpha = \pi/2$ compatible with the drag coefficient classically given for a flat obstacle in the inertial regime (high Froude number). Figure 2 also reports the force normalized by the purely hydrostatic force $\frac{1}{2}\rho gh^2 \cos \theta$. It is interesting to consider the asymptotic prediction for this ratio when θ tends towards θ_{min} (i.e. Fr tends to zero). According to the analytical model, this ratio should scale as $[k + (H/h)(2 + H/h)(1/\cos \theta_{min})]$. With $k = 1$ and $H/h = 1$, it gives a value of $[1 + 3/\cos \theta_{min}]$. This ratio is then close to 4 ($\cos \theta_{min} \approx 1$) as found in discrete simulations and shown in Figure 2. In the following section, we propose to use the analytical model for dry snow and we provide the model parameters adapted to dry snow.

Outlook for 2D snow flows

In this section we propose to use Equation 4 for dry snow flows and we try to define the parameters characterizing the behaviour of dry snow. Cemagref has designed a 10-m-long and 20-cm-wide channel at the Col du Lac Blanc (Alpe d'Huez, France), which is described in detail by Bouchet and others (2003; 2004). Recent investigations on flows of dense and dry snow (for

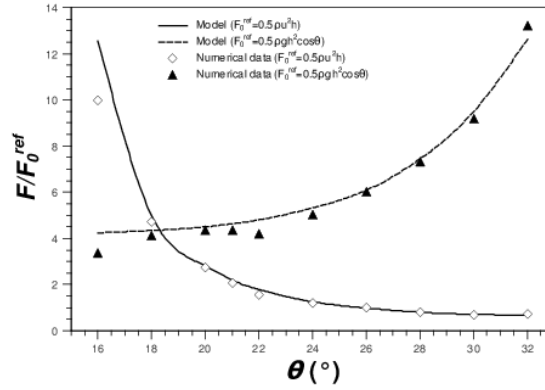


Fig. 2. Granular flows: prediction of the model for the rescaled force $F/(0.5\rho u^2 h)$, respectively $F/(0.5\rho g h^2 \cos \theta)$, versus the slope θ , compared to data from numerical discrete simulations. The following parameters were used: $\beta = 5/4$, $k = 1$, $\kappa = 0.31$ ($e = 0.5$), $\theta_{min} = 14^\circ$, $\theta_{max} = 24^\circ$ (Faug and others, 2009).

117 $T < 0^\circ\text{C}$) down this 10-m-long flume showed that snow exhibits some properties similar to those of granular flows (Rognon,
 118 2006; Rognon and others, 2008). Dry and dense snow is a polydisperse granular material. Similarly to granular flows, there
 119 exists a minimum angle below which the flow is stopped and a maximum angle above which flows are accelerating along the
 120 channel. Steady and uniform flows are only possible between these two angles. Typical values were derived from experimental
 121 investigations at Col du Lac Blanc pass for dense and dry snow (Rognon and others, 2008): $\theta_{min} = 33^\circ$ and $\theta_{max} = 42^\circ$.
 122 These angles were obtained in a narrow channel with typical flow depths around 10 cm. Even if these angles are likely to be
 123 influenced by wall effects (whose extent is still an open question), we will use these angles for snow in the following. The flow
 124 is divided into two layers: a highly sheared layer of snow grains (typically 1 mm in size) at the base surmounted by a low shear
 125 layer of aggregates (with a maximum size close to the flow depth). This flow configuration results in a typical velocity profile
 126 close to velocity profiles obtained from discrete numerical simulations on bi-disperse granular flows. A value of β close to one
 127 is then reasonable for snow flows.

128 The friction law for snow flows is still an open question. However, an effective friction law corresponding to a Voellmy model
 129 could be fitted on data from the Lac Blanc chute (Rognon, 2006): $\mu = \mu_s + (g/\xi)Fr^2$. The Voellmy model is classically used
 130 in snow avalanche engineering applications (Bartelt and others, 1999). μ_s is a dry friction coefficient and ξ is a turbulent
 131 coefficient. Both parameters (μ_s, ξ) were fitted corresponding to various choices. The best fit was obtained for $\mu = 0.61$, i.e
 132 $\theta_s = \arctan(\mu_s) = 31.2^\circ$, and $\xi = 1050 \text{ m s}^{-2}$, which are values encountered for snow avalanches (Salm and others, 1990).
 133 With respect to the limit angle $\theta_{min} = 33^\circ$, which would imply $\mu = \tan \theta_{min}$, a value of $\xi = 1400 \text{ m s}^{-2}$ was obtained.

134 We assume that the empirical laws derived from investigations on granular flows to estimate the free-surface angle α_{sl} , the
 135 dead zone angle α_{zm} , and consequently the deflection angle α , as well as the friction μ_{zm} , still hold for our granular-like snow

flows. A value of $k = 1$ is chosen. Here, κ is calculated using a very low value for e that is more compatible with the properties of snow material: $e = 0.1$ gives $\kappa = 0.57$.

The angle α is assumed to be equal to:

$$\alpha = \frac{\alpha_{zm} + \alpha_{sl}}{2}, \quad (7)$$

where α_{sl} is defined by equations 5 and 6.

In steady and uniform flow conditions, or within a flow regime for which the effect of acceleration terms (time-derivative terms in momentum conservation) can be neglected, we have the following equation corresponding to equilibrium between gravity and friction forces ($\tan \theta = \mu$):

$$\theta = \arctan \left(\tan \theta_{min} + \frac{g}{\xi} Fr^2 \right). \quad (8)$$

This last equation allows us to eliminate the slope θ in equations 7 and 4 and to express the rescaled force $F/(0.5\rho u^2 h)$ as a function of the incoming Froude number Fr without the prior knowledge of the slope (information included in the Froude number). This raises the question whether we may extrapolate the predictions of our analytical model to slope inclinations lower than θ_{min} in transient conditions towards stopping. The answer to this question is positive if the system is able to reach an equilibrium state for which the assumption $\tan \theta = \mu$ still holds. This assumption seems to be reasonable in the case of decelerating flows evolving towards stopping (tails of avalanches) for which the effect of time-derivative terms is expected to be negligible. We will use this assumption in the following when applying equations established for the steady regime to full-scale avalanche flows.

It is important to highlight that the rescaled force $F/(0.5\rho u^2 h)$ depends on the ratio H/h . Indeed, attempts to find a relation between this ratio and Fr are meaningful only if H/h remains constant. Figure 3 gives the prediction of the analytical model in terms of the rescaled force versus the Froude number for different values of H/h , θ_{min} and θ_{max} , keeping $(\theta_{max} - \theta_{min})$ constant. The curves show that the results are not very sensitive to the value of θ_{min} . The results are rather influenced by the geometry corresponding to a varying ratio H/h . We also reported the rescaled force $F/(0.5\rho g h^2 \cos \theta)$ (for the same set of parameters) to show the prediction of the analytical model at very low Froude numbers in order to highlight the transition towards the hydrostatic regime.

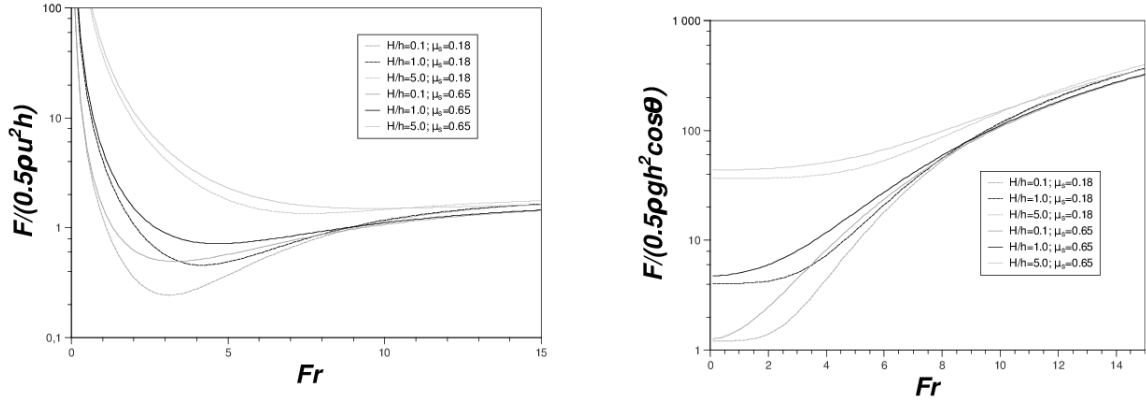


Fig. 3. Snow flows: Prediction of the analytical model for the rescaled forces $F/(0.5\rho u^2 h)$ (left) and $F/(0.5\rho g h^2 \cos \theta)$ (right) versus the Froude number at different ratios H/h (0.1, 1 and 5). The following parameters were used: $\beta = 1$, $k = 1$, $\kappa = 0.57$ ($e = 0.1$) and $\xi = 1000 \text{ ms}^{-2}$. Predictions are given for two pairs $(\theta_{min}, \theta_{max})$ keeping $(\theta_{max} - \theta_{min})$ constant equal to 9° : (i) $[33^\circ; 42^\circ]$ ($\mu_s = 0.65$) and (ii) $[10^\circ; 19^\circ]$ ($\mu_s = 0.18$).

158 Outlook for 3D effects with lateral fluxes

159 General framework equations in 3D geometry

160 We propose a modification of Equation 4, taking into account 3D effects corresponding to lateral fluxes and to the modification
 161 of the shape of the dead zone by these lateral fluxes. The resulting analytical model is a simplified model that does not take into
 162 account the entire effect of flow spreading, which would require fully 3D numerical models (e.g. Naaïm and others, 2004). Here,
 163 we consider the thickness h^L and the depth-averaged velocity u^L that correspond to the lateral fluxes around the obstacle.
 164 Figure 4a gives a top view of the flow configuration in 3D geometry. A dead zone is formed upstream of the obstacle. The base
 165 of the dead zone in the plane (x, y) is triangular and is characterized by the angle γ .

166 Compared to the 2D geometry, the angles α_{zm} , α_{sl} and α depend on the transverse position y along the width ℓ of the
 167 obstacle as depicted in Figure 4b. As a first approximation and in want of a well-documented experimental evidence about the
 168 shape of the dead zone, we can assume a simple triangular shape for the dead zone in a plane normal to the z -axis direction
 169 (Fig. 4a), and the length $L(y)$ at location y can be expressed as $L(y) = c + dy$. The boundary conditions requires: $0 = c + d(\ell/2)$
 170 (edge of the obstacle) and $H/\tan(\theta - \theta_{min}) = c$ (centre of the obstacle), which leads to:

$$L(y) = \frac{H}{\tan(\theta - \theta_{min})} \left(1 - \frac{2}{\ell}y\right). \quad (9)$$

171 The dead zone angle at position y is $\alpha_{zm}(y) = \arctan(H/L(y))$. The free-surface angle is assumed to vary linearly between
 172 $\pi/2$ (edge of the obstacle) and the value of α_{sl} in 2D given by Equations 5 and 6, which gives: $\alpha_{sl}(y) = 2(\pi/2 - \alpha_{sl})(y/\ell) + \alpha_{sl}$.

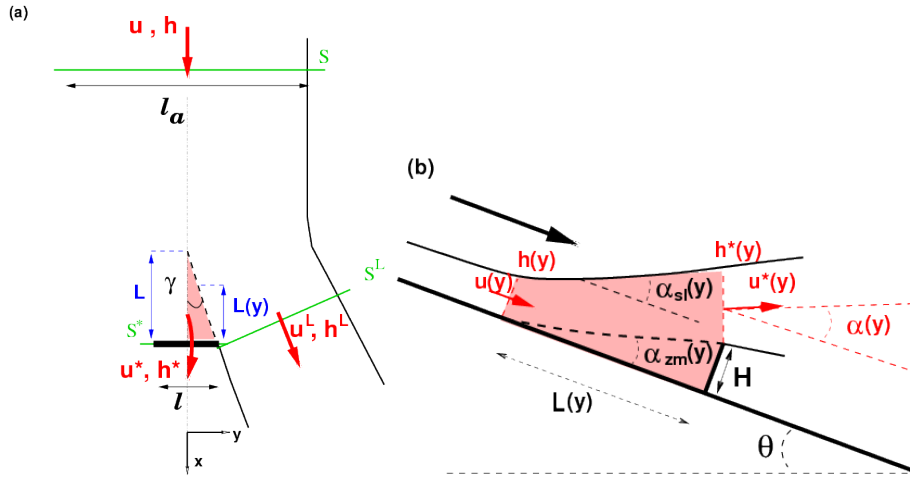


Fig. 4. (a) Top view of the flow and dead zone (pink colored area) in 3D geometry. γ is the mean angle of the dead zone in the plane (x, y) . h and u are the thickness and the depth averaged velocity outside of the zone of influence of the obstacle. l_a is the width of the incoming flow. h^* and u^* are the thickness and the depth averaged velocity at the top of the dam. ℓ is the width of the obstacle. h^L and u^L are the mean thickness and the depth averaged velocity in the flow branch corresponding to lateral fluxes. L is the length of the dead zone at the centre (in y -axis direction) of the obstacle, and $L(y)$ is the length of the dead zone at a given position y . Note that h^* , u^* , h^L and u^L are mean values in sections S^* (overflow) and S^L (lateral fluxes). Due to the symmetry of the problem, we only show one lateral flux. (b) Side view at a position y of the flow overflowing the obstacle. We use a notation similar to the one of the 2D configuration given in Figure 1 but here the variables depend on the position y .

The mean angle of deflection is then given by: $\alpha(y) = [\alpha_{zm}(y) + \alpha_{sl}(y)]/2$. The dependence of these angles on y complicates the calculation of the momentum balance (3D integrals).

We assume that the basal friction is proportional to the weight in the z -axis direction. The total basal force is caused by the basal friction below the dead zone and the sum of the weights of the dead zone and the fluid above the dead zone:

$\bar{F}_w - \bar{F}_f = (\sin \theta - \mu_{zm} \cos \theta) \rho g \bar{V}$. \bar{V} is the control volume in 3D geometry (between sections S , S^* and S^L in Fig. 4):

$$\bar{V} = \frac{2}{\ell} \int_0^{\ell/2} \left[\frac{1}{2} L(y) \left(H + h \left(1 + \frac{\delta_h(y)}{\cos(\alpha(y))} \right) \right) - \frac{1}{2} h^2 \delta_h(y) \tan(\alpha(y)) \right] dy. \quad (10)$$

Equation 10 can be solved numerically. This work is in progress and not achieved yet. In the following section, we propose some rough approximations to simplify the model in order to be able to provide an analytical solution under restrictive conditions.

Simplified analytical model for 3D snow flows

For the sake of simplicity and to provide an analytical model, we here consider the mean values over the obstacle width:

$$\bar{\alpha}_{sl} = \frac{1}{(\ell/2)} \int_0^{\ell/2} \alpha_{sl}(y) dy = \frac{\pi}{4} + \frac{\alpha_{sl}}{2}, \quad (11)$$

$$\bar{\alpha}_{zm} = \frac{1}{(\ell/2)} \int_0^{\ell/2} \alpha_{zm}(y) dy, \quad (12)$$

$$\bar{\alpha} = \frac{\bar{\alpha}_{sl} + \bar{\alpha}_{zm}}{2}. \quad (13)$$

183 One parameter is introduced: $r = \ell_a/\ell$ is the ratio between the width of the incoming avalanche flow ℓ_a and the width of the
 184 obstacle ℓ . Similarly to the 2D case, we can apply the momentum balance over the control volume \bar{V} . We roughly simplified
 185 the calculation by taking the mean values over the obstacle width (given by Equations 11, 12 and 13) out of the 3D integrals
 186 which gives:

$$\left(\frac{F/\ell_a}{\frac{1}{2}\rho u^2 h} \right)_u = 2\beta \left[1 - \frac{1}{r} \bar{\delta}_u^2 \bar{\delta}_h \cos \bar{\alpha} - \left(1 - \frac{1}{r} \bar{\delta}_u \bar{\delta}_h \right) \bar{\delta}_u^L \cos \gamma \right], \quad (14)$$

$$\left(\frac{F/\ell_a}{\frac{1}{2}\rho u^2 h} \right)_{h+w-f} = \frac{1}{Fr^2} \left[k + \frac{1}{4r} (\cos \theta + (\tan \theta_{min}) \sin \theta) \left(\frac{H}{h} + 1 + \frac{\bar{\delta}_h}{\cos \bar{\alpha}} \right) \frac{H}{h} \right]. \quad (15)$$

187 We split the resulting ratio $(F/\ell_a)/(0.5\rho u^2 h)$ into two parts: Equation 14 is the contribution due to the dynamic force
 188 and Equation 15 is the contribution of the sum of the incoming pressure force, the weight and the basal friction force. Three
 189 parameters have to be quantified: the mean ratio of velocities at the centre, $\bar{\delta}_u$, the mean ratio of the flow depths, $\bar{\delta}_h$, and the
 190 mean ratio of velocities on both lateral sides $\bar{\delta}_u^L$. The mean lateral flow depths ratio $\bar{\delta}_h^L = \bar{h}^L/h$, where \bar{h}^L is the thickness of
 191 lateral fluxes, is determined by the conservation of the mass flow rate: $1 - \bar{\delta}_u \bar{\delta}_h (1/r) = \bar{\delta}_u^L \bar{\delta}_h^L [1 - (1/r)]$.

192 We use the following assumptions to derive simple empirical laws for these quantities. The mean ratio of velocities at the
 193 centre is calculated from the mean angle $\bar{\alpha}$: $\bar{\delta}_u = 1 - \kappa \bar{\alpha}$, similarly to the 2D geometry. The mean ratio of flow depths at
 194 the centre is assumed to be close to 1, which corresponds to the assumption that the typical size of the overflowing flow
 195 is close to the typical size of the undisturbed flow. This is almost true for the case of 2D granular flows (Faug and others,
 196 2009) but it remains an assumption for 3D flows, in want of existing well-documented experimental data. The mean ratio of
 197 velocities on both lateral sides is assumed to depend on the mean angle γ (by analogy with the definition of $\bar{\delta}_u$ according
 198 to $\bar{\alpha}$): $\bar{\delta}_h^L = 1 - \kappa^L \gamma$, where we simply assume a value of κ^L equal to $\kappa = (1 - e)/(\pi/2)$. This latter assumption is argued
 199 by the fact that the restitution coefficients are similar for collisions in the planes (x, z) and (x, y) . The angle γ is defined by
 200 $\gamma = \arctan[(\ell/2)/L]$ (Fig. 4a).

MODEL PREDICTIONS COMPARED TO FULL-SCALE FIELD DATA

Only few data corresponding to the geometry considered here are available in literature. Three large scale avalanche test sites provide some promising data. First, the Lautaret avalanche site in France and particularly the track equipped with a flat obstacle of surface equal to 1 m^2 (Thibert and others, 2008) is of interest even if the geometry is complex due to lateral fluxes, jet and spreading effects. Second, the Vallée de la Sionne test site in Switzerland provides well documented data but only small size obstacles compared to the width of the avalanche flow (pylons or flat obstacle) are investigated (Sovilla and others, 2008a,b) that do not fit into the framework of our analytical model. Third, the Ryggfonn test site in Norway is of interest because it is close to the geometry of our analytical model but it still suffers of a lack of flow-depth data implying an unknown Froude number (Faug and others, 2008b) and the dam is not normal to the incident flow.

The full-scale avalanche flows are transient flows that, a priori, are not compatible with our 3D analytical model for which we assume a steady state. As discussed above, we assume that the effects of time-derivative terms in the equations of motion are weak so that a quasi-steady state is reached at each time t and that the basal friction μ is close to $\tan \theta$. These assumptions allow us to express θ as a function of the incoming Froude number, where $\mu = \tan \theta_{min} + (g/\xi)Fr^2$ (Voellmy friction law). Then we can describe variations of $(F/\ell_a)/(0.5\rho u^2 h)$ with the Froude number.

We propose to compare the predictions of the simplified 3D model to the measurements from the Lautaret test site. One of the avalanche paths is equipped with a 1 m^2 plate with pressure sensors. The avalanche site, the instruments and the procedure are presented in detail by Thibert and others (2008). In Figure 5, we compare the prediction from Equations 14 and 15 to the data of two avalanches from the Lautaret test site, February 15, 2007 (Thibert and others, 2008) and March 26, 2008 (Baroudi and Thibert, 2009). Both avalanches were released in cold and dry snow conditions. Furthermore, the time-derivative terms were estimated and shown to be negligible for the 2007 avalanche in the decelerating flow-phase (see Thibert and others, 2008, Fig. 11). We only consider data obtained in the decelerating flow-phase of the avalanche. The following set of parameters was used: $\beta = 1$, $k = 1$, $H/h = 1$, $\theta_{min} = 33^\circ$, $\theta_{max} = 42^\circ$, $\xi = 1000 \text{ ms}^{-2}$ and $\kappa = \kappa^L = (1 - e)/(\pi/2)$ with $e = 0.1$. We used $r = 7$ for the February 15, 2007 avalanche and $r = 3$ for the March 26, 2008 avalanche. These values of r are compatible with the field observations when considering the movies of the avalanches. The 2007 avalanche was substantially larger than the 2008 avalanche. In spite of many assumptions, the prediction of our 3D simplified model is in good agreement with the field data (full black line in Fig. 5). We also plot both contributions to the total force in Figure 5: one contribution from the dynamic force (equation 14) and the other contribution (equation 15) corresponding to the sum of the hydrostatic force, the gravity force and the basal friction. The graphs clearly show that the increase of the total rescaled force $(F/\ell_a)/(0.5\rho u^2 h)$ at low values of the Froude numbers (Fr around 1) is mainly due to the contribution corresponding to the sum of the hydrostatic force, the gravity force and the basal friction. Note that the increase of the rescaled force $(F/\ell_a)/(0.5\rho u^2 h)$ does not mean

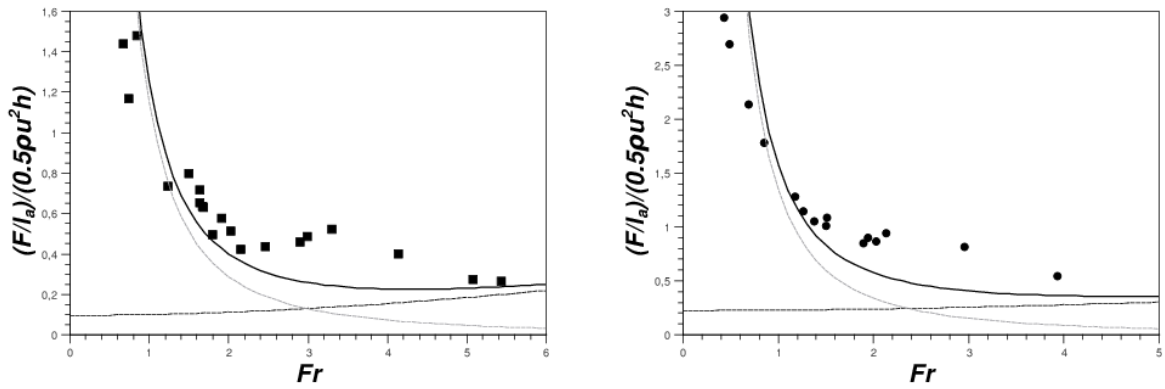


Fig. 5. Avalanche flows at Lautaret: predictions of the analytical model compared to the measured data (full black line). The following parameters were used: $\beta = 1$, $k = 1$, $H/h = 1$, $\theta_{min}=33^\circ$, $\theta_{max}=42^\circ$, $\xi = 1000 \text{ ms}^{-2}$ and $\kappa = \kappa^L = (1 - e)/(\pi/2)$ with $e = 0.1$. The gray dashed line shows the contribution from the sum of the hydrostatic force, the gravity force and the basal friction force (Eq. 15). The black dashed line shows the contribution from the incoming dynamic force (Eq. 14). Left-side graph: the February 15, 2007 avalanche (Thibert and others, 2008) with $r = 7$. Right-side graph: the March 26, 2008 avalanche (Baroudi and Thibert, 2009) with $r = 3$.

231 that the corresponding force F becomes the design force. For both avalanches investigated here, the force is maximal when
 232 the Froude number is higher (close to 4 – 5) and in an engineering project, this maximal force would be the design force. By
 233 this graph we intend to demonstrate that when the avalanche comes to rest, the contributions from forces due to hydrostatic
 234 effect, weight and friction become dominant. This effect is particularly important in the run-out areas of snow avalanches.
 235 The simplified analytical model presented here has to be validated on more data. However, this analytical model can provide
 236 the ingredients for a practically applicable approximation of the total force exerted on obstacles by snow avalanches. When the
 237 design reference Froude number is low and large stagnant zones are likely to be formed upstream of the protection structure,
 238 it is crucial to check whether the resulting force F calculated from the proposed model is greater than the force calculated
 239 from traditional engineering methods (Salm and others, 1990).

240 DISCUSSION AND CONCLUSION

241 The present paper deals with an analytical model to estimate the force on flat obstacles when a stagnant zone is formed
 242 upstream of the obstacle. We first presented a 2D analytical model describing gravity-driven flows overflowing a wall normal
 243 to the ground. Three important parameters were needed to close the model: the friction (μ_{zm}) between the dead zone and
 244 the ground, the coefficient of velocity reduction (κ) and the angle of the free-surface upstream of the wall (α). The analytical
 245 2D model and the empirical laws proposed to quantify these parameters were validated by discrete particle simulations of
 246 granular flows down an inclined slope. Then we provided the needed parameters to use this analytical model for 2D flows of

247 dry snow, which allowed us to quantify the effect of the stagnant zone on the resulting force, particularly at low incoming
248 Froude numbers and to provide a tool to estimate the force on large catching dams with no (or little) lateral overflows. This
249 2D analytical model was then extended to a 3D configuration in order to predict the force when lateral fluxes occur. The
250 analytical model was stated for steady flow conditions but we believe that it is also suitable for transient flows for which
251 the effect of time-derivative terms can be neglected. This is typically the case for snow avalanches in the decelerating phase
252 (before the final standhill). We compared the prediction of our analytical model to the field data available at Lautaret. We only
253 used the data in the quasi-steady state corresponding to the decelerating phase (see Thibert and others, 2008, Fig. 11). The
254 analytical predictions are in good agreement with field observations within the experimental uncertainty. More validation of
255 the analytical model is needed (not only on the force but also on geometrical data such as the deflecting angles and the shape
256 of the stagnant zone). Let us stress the fact that the time-dependency of avalanche flows has not been considered in this paper,
257 which means that our proposed model is not devoted to predict acting transient forces on obstacles induced by the avalanche
258 front. Further work is needed to analyse these transient forces and the related fluctuations. Furthermore, a sensitivity analysis
259 of the model with respect to all the parameters should be performed in the future before being able to propose the sketched
260 methodology to serve as conceptional base for a practitioners' recipe for estimating design forces for structures that can be hit
261 by avalanches. The parameters combinations for various avalanche types, similar to the ones by Salm and others (1990) for
262 the estimation of avalanche run-out lengths and velocities with the Voellmy friction law, could be used to carry out this task.
263 However, we recommend to investigate the force at low Froude numbers, when stagnant zone mechanisms are likely to occur,
264 in a more fundamental level.

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